



ECOLE  
POLYTECHNIQUE  
DE BRUXELLES

ULB

UNIVERSITÉ LIBRE DE BRUXELLES

# Experimental and numerical analysis of a Pump as Turbine (PaT) in micro Pumped Hydro Energy Storage ( $\mu$ -PHES)

A thesis submitted to obtain the academic degree of  
Doctor of Engineering Sciences and Technology

**Alessandro Morabito**

Supervisor  
Professor Patrick Hendrick

Departement  
Aero-Thermo-Mechanics

Academic year  
2020 - 2021



# Abstract

In the last decade, the power generation mix and the energy markets have been affected by the growing development of distributed and renewable energy sources. Nevertheless, a significant drawback of solar and wind energy is their intermittent and weather-dependent production, which often leads to a mismatch between renewable energy production and its use. Thus, the need for energy storage is recently emerging and becoming more relevant in this era of the energy transition.

Among several technologies, today, pumped hydro energy storage (PHES) represents the largest share of the energy storage systems in the world. However, possible new investors, who might be attracted by potential profit in PHES, are repelled by the long payback period and the scarcity of adequate site topology for such power plants. Relevant design decisions can be taken to reduce the costs and improve the performance or to escape the PHES topographical requirements. For this reason, the first part of this PhD thesis reviews and provides potential assessments of some unconventional PHES systems, applied in synergy with existing infrastructures. Such is the standpoint of micro facilities near waterway locks, or underground cavities used as lower reservoirs (UPSH), or the use of pump-turbines at variable geometry to cope with fluctuating loads. Moreover, important information on PHES in micro-scale is largely missing and their potential in distributed energy systems still needs to be unveiled. In the attempt to fill this gap, this thesis provides a techno-economic overview of the design and characterization of a first-of-its-kind PHES micro facility. In micro-scales hydropower projects, the initial capital cost of a conventional hydroelectric unit is hard to be determined and often economically prohibitive. Interestingly, in order to cut the total capital investment, the micro-PHES prototype runs with a single centrifugal pump for both pumping and generating phases and exploits existing stormwater reservoirs. The variable speed regulation is also implemented and it allows the pump to constantly operate at the maximum hydraulic efficiency in order to deal with load variations. In the same way, the pump working in reverse, namely pump as turbine (PaT), runs at the most suitable speed and it keeps a high efficiency over a wide load range. In addition, the analysis of the techno-economic parameters for such a system provides an important dataset for micro-PHES feasibility breakdown.

PaTs are a legitimate cost-effective option in micro hydropower but an universal performance prediction does not exist. Their hydraulic efficiency can possibly shift from the higher efficiency of traditional hydraulic turbines. Nowadays, these reasons restrict PaTs exploitation. In this thesis, a multivariate regression method is applied to the CFD results to build a surrogate model of the PaT hydraulic characteristics as a function of the cutwater geometrical modifications. Based on this model, an optimization problem is solved to identify the most advantageous geometrical asset of the PaT cutwater to maximize the hydraulic efficiency. The presented methodology and design optimization of the cutwater in PaTs, which are extremely suited to our current energy generation needs, provides a unique and much sought guide to its performance, improvements, and adaptation to hydropower.

# Contents

<b>Abstract</b>	<b>i</b>
<b>Acknowledgments</b>	<b>iii</b>
<b>Contents</b>	<b>vii</b>
<b>List of Figures</b>	<b>xiii</b>
<b>List of Tables</b>	<b>xvi</b>
<b>Nomenclature</b>	<b>xvii</b>
<b>1 Introduction</b>	<b>1</b>
1.1 Context and motivation . . . . .	1
1.2 Thesis objectives and strategy . . . . .	3
1.3 Thesis structure . . . . .	4
1.4 Publications . . . . .	6
<b>2 Literature review and state-of-the-art</b>	<b>7</b>
2.1 Introduction . . . . .	7
2.2 Pumped hydro energy storage system (PHES) . . . . .	9
2.2.1 Traditional hydro power station . . . . .	9
2.2.2 Unconventional PHES . . . . .	11
2.2.3 Underground Pumped Storage Hydroelectricity . . . . .	12
2.3 Turbomachinery options for hydropower and PHES . . . . .	14
2.3.1 Characteristic curves . . . . .	14
2.3.2 Hydraulic turbines for hydropower and PHES . . . . .	16
2.3.3 Variable speed regulation . . . . .	22
2.4 The Deriaz pump-turbine . . . . .	24
2.4.1 Variable geometry regulation . . . . .	25
2.4.2 Deriaz in PHES plants . . . . .	28
2.5 Pump as Turbine (PaT) . . . . .	31
2.5.1 Variable speed regulation applied to PaTs . . . . .	37
2.5.2 PaT efficiency improvements . . . . .	38
<b>3 Definition of a framework</b>	<b>41</b>
3.1 Introduction . . . . .	41
3.2 Micro PHES design: the Tucuruí case study . . . . .	45
3.2.1 Introduction to the project . . . . .	45
3.2.2 Characteristics of Tucuruí locks . . . . .	46
3.2.3 The facility . . . . .	47

3.2.4	The selection of the machines . . . . .	48
3.2.5	Operation scheme . . . . .	49
3.2.6	Energy payback time . . . . .	50
3.2.7	Perspectives for the micro PHES design in tucuruí . . . . .	51
3.3	UPSH case studies in Belgium . . . . .	52
3.3.1	Towards PHES solutions in Belgium . . . . .	52
3.3.2	The slate quarry of Martelange case study . . . . .	56
3.3.3	The coal mine of Péronnes-lez-Binche case study . . . . .	63
3.3.4	UPSH in Belgium - Discussion . . . . .	69
3.4	Deriaz pump-turbine . . . . .	72
3.4.1	Deriaz hydraulic model . . . . .	72
3.4.2	Selected model case . . . . .	76
3.4.3	Numerical analysis . . . . .	77
3.4.4	Final remarks on Deriaz pump-turbine . . . . .	85
3.5	Chapter conclusions . . . . .	87
<b>4</b>	<b>Micro-PHES prototype</b>	<b>91</b>
4.1	Introduction . . . . .	91
4.1.1	Goals of the analysis . . . . .	92
4.2	Micro-PHES in the " <i>Quartier Negundo</i> " . . . . .	94
4.2.1	Pipeline and pressure losses . . . . .	98
4.2.2	Turbomachinery selection . . . . .	99
4.2.3	Set-up and instrumentation . . . . .	105
4.2.4	Error measurement propagation . . . . .	108
4.2.5	Experimental methodology . . . . .	110
4.3	Results . . . . .	115
4.3.1	Experimental characterisation . . . . .	115
4.3.2	Variable rotational speed efficiency gain . . . . .	118
4.3.3	A simulated day of the micro-PHES . . . . .	121
4.4	Cost-benefit analysis . . . . .	123
4.4.1	Sector description of smart-grid business model . . . . .	123
4.4.2	$\mu$ -PHES prototype in the <i>Quartier Negundo</i> . . . . .	130
4.4.3	$\mu$ -PHES case study . . . . .	133
4.4.4	Economic-evaluation methods . . . . .	134
4.4.5	Analysis results . . . . .	136
4.5	Conclusions on micro PHES solution . . . . .	140
<b>5</b>	<b>Numerical investigation</b>	<b>141</b>
5.1	Introduction . . . . .	141
5.2	Problem statement . . . . .	142
5.3	Methodology . . . . .	143
5.3.1	Experimental setup . . . . .	143
5.3.2	Hydraulic domain modelling . . . . .	144
5.3.3	Mesh generation . . . . .	151

---

5.3.4	Numerical Modelling . . . . .	155
5.3.5	Multivariate regression model . . . . .	157
5.3.6	Optimization problem . . . . .	159
5.4	Results . . . . .	160
5.4.1	Validation of PaT numerical simulations . . . . .	160
5.4.2	Performance evaluation . . . . .	162
5.4.3	Surrogate model of the PaT hydraulic performance . . . . .	168
5.4.4	Optimal cutwater design . . . . .	171
5.4.5	Unsteady verification of the PaT optimum . . . . .	178
5.4.6	Development of speed adjustment . . . . .	183
5.4.7	Pump performance . . . . .	185
5.4.8	Unsteady verification of the pump performance in $\Lambda_{opt}$ . . . . .	191
5.5	Chapter conclusions . . . . .	194
<b>6</b>	<b>Conclusions and perspectives</b>	<b>197</b>
6.1	Achievements . . . . .	197
6.1.1	Review of the objectives . . . . .	197
6.1.2	Work novelty . . . . .	200
6.2	Future work and perspectives . . . . .	201
6.2.1	Variable geometry Deriaz pump-turbine . . . . .	201
6.2.2	UPSH . . . . .	202
6.2.3	$\mu$ -PHES design recommendations . . . . .	203
6.2.4	Optimal cutwater design finalisation in PaT . . . . .	205
	<b>Appendices</b>	<b>207</b>
<b>A</b>	<b>Hydraulic design of a diagonal pump</b>	<b>209</b>
A.1	Introduction . . . . .	209
A.1.1	Scaling laws . . . . .	210
A.1.2	Impeller inlet . . . . .	213
A.1.3	Impeller outlet . . . . .	214
A.1.4	Blade profile . . . . .	217
<b>B</b>	<b>Fundamental notes on cavitation</b>	<b>221</b>
	<b>Bibliography</b>	<b>225</b>

# List of Figures

1.1	Objectives of the PhD thesis. . . . .	5
2.1	World growth in renewable electricity generation . . . . .	8
2.2	The operation principle of a PHES: it stores energy in pump operation and it generates energy in turbine operation. . . . .	10
2.3	Seawater-based pumped storage plant in Okinawa . . . . .	11
2.4	Hydraulic gravity storages (HGS) . . . . .	12
2.5	Simplified schematic diagram of an UPSH facility . . . . .	13
2.6	Representation of variable head during pumping water in a tall and narrow tank . . . . .	15
2.7	Representation of variable head during turbine generation . . . . .	15
2.8	Types of hydraulic turbines in their application area Q-H . . . . .	17
2.9	Turbine runners . . . . .	18
2.10	Centrifugal pump in normal and inverse mode. . . . .	20
2.11	Pump characteristic curves for different rotational speeds . . . . .	23
2.12	Internal view of Deriaz pump-turbine runner . . . . .	25
2.13	Turbine runners . . . . .	25
2.14	Diagram of hydraulic turbines . . . . .	26
2.15	Shapes of Deriaz pump-turbine casing . . . . .	27
2.16	Deriaz runner normally open and closed . . . . .	27
2.17	Normalised efficiency and discharge of Deriaz-pump-turbine . . . . .	28
2.18	Radial view of Deriaz turbine guide vanes . . . . .	30
2.19	Pump four quadrant characteristics . . . . .	32
2.20	Representations of the operating regimes in four quadrants . . . . .	32
2.21	Reliability impact of operation away from BEP . . . . .	33
2.22	Types of hydraulic turbines and PaT for micro-hydropower . . . . .	34
2.23	PaT discharge and head ratios . . . . .	36
2.24	Qualitative representation of variable speed adjustments . . . . .	37
2.25	PaT hydraulic efficiency comparison . . . . .	39
3.1	"Demand will follow generation" vision . . . . .	43
3.2	Mine and quarry exploitation in the Walloon region . . . . .	44
3.3	Overview of the Tucuruí hydropower dam and its locks. . . . .	46
3.4	Schematic representation of the Tucuruí locks and of the installation. . . . .	47
3.5	Pump mode system operation in Tucuruí micro PHES. . . . .	48
3.6	Power production in the hybrid system. . . . .	49
3.7	Hybrid system operation scheme. . . . .	49
3.8	Comparison of energy alternatives payback. . . . .	51
3.9	Belgian electricity generation by source, 1990-2019 . . . . .	52
3.10	Representation of Coo-Trois-Ponts PHES plant and its location . . . . .	55

3.11	The slate mine on the Martelange site . . . . .	56
3.12	View of Martelange location and topography . . . . .	57
3.13	Contingencies costs in percentage of the plant total cost . . . . .	59
3.14	Discounted cash flow evolution and NPV of the Martelange case study. . . . .	60
3.15	Evolution of the gross head, hydraulic efficiency and power for three turbomachinery configurations . . . . .	62
3.16	PaTs performance estimation for Martelange case study . . . . .	63
3.17	Scheme of the coal mine of Péronnes-lez-Binche . . . . .	64
3.18	Schematic representation of Francis turbines in series . . . . .	65
3.19	SIMSEN components to simulate the Péronnes-lez-Binche power plant. . . . .	67
3.20	Gross head, available head and discharge of each Francis . . . . .	68
3.21	Operating sequence of the three Francis turbines in series . . . . .	68
3.22	Gross head, available head and discharge of each Francis turbine . . . . .	69
3.23	Overall cross-section of a Deriaz pump turbine in Naussac II power- house. . . . .	74
3.24	Deriaz CFD research domain . . . . .	77
3.25	Flow streamlines at the pump inlet and flow speed distribution at the leading edge. . . . .	81
3.26	Head and efficiency of Deriaz pump-turbine in pump mode . . . . .	82
3.27	Experimental and numerical data of Deriaz turbine . . . . .	84
3.28	Efficiency map of Deriaz turbine at $\delta\beta + 5$ blade angle . . . . .	84
3.29	Hill chart of Deriaz turbine at fixed runner blade angle . . . . .	85
3.30	Torque map of Deriaz turbine at fixed runner blade angle . . . . .	85
3.31	Types of hydraulic turbines and Deriaz turbine in their application area Q-H for $\mu$ -hydropower . . . . .	86
4.1	Schematic representation of the implemented $\mu$ -PHES, integrated in its micro smart grid. . . . .	93
4.2	Overview of the <i>Quartier Negundo</i> in Froyennes, Tournai . . . . .	94
4.3	Interconnection scheme of <i>Quartier Negundo</i> smart grid. . . . .	95
4.4	View of the upper reservoir in <i>Quartier Negundo</i> . . . . .	96
4.5	Internal view of the lower reservoir of the $\mu$ -PHES. . . . .	97
4.6	Traversal view of the reservoirs and the technical room in <i>Quartier Negundo</i> . . . . .	97
4.7	Photo of protection grid at the pipeline inlet at the upper reservoir . . . . .	98
4.8	Estimation of $h$ and $q$ ratios by different methods . . . . .	100
4.9	Regression for head ratio $h$ and discharge ratio $q$ of available PaT experimental data . . . . .	101
4.10	Decision tree for PaT selection. . . . .	102
4.11	Sectional view and photo of the pump/PaT of the $\mu$ -PHES . . . . .	103
4.12	Powerhouse in $\mu$ -PHES of the <i>Quartier Negundo</i> . . . . .	104
4.13	Schematic view of the micro-hydropower station and location of the measurement devices . . . . .	105
4.14	Pressure transmitted manifold practise . . . . .	106



4.15	Measuring instrumentation . . . . .	107
4.16	Control instrumentation . . . . .	107
4.17	Error propagation in function of the dimensionless discharge number. . . . .	109
4.18	Schematic layout of the set-up . . . . .	110
4.19	Pressure measurements for sensor A, B and C facing PaT run-up, opening valve, normal operation, closing valve and shut-down. . . . .	111
4.20	Human-machine interface developed on the LabView for real time control and analysis. . . . .	112
4.21	Experimental results of the pump characteristics and its hydraulic efficiency. . . . .	115
4.22	Experimental results of the pump and its power consumption. . . . .	116
4.23	Experimental results of PaT characteristics in Q-H plot limited by the runaway curve. . . . .	116
4.24	Experimental results on PaT efficiency over the variable head, $H$ . . . . .	117
4.25	Electrical efficiency and torque over the variation of the rotational speed for turbine and pump modes. . . . .	117
4.26	Gain in hydraulic efficiency in pump mode by using variable speed over the variation of the head . . . . .	119
4.27	$\eta_{PAT}$ normalized to its maximum over H at different speed regimes . . . . .	120
4.28	Gain in hydraulic efficiency in turbine mode by using variable rotational speed over the variation of the head. . . . .	120
4.29	Absorbed and produced power by the storage system according to the load profile and RES production . . . . .	122
4.30	Imbalance power (SI) and prices of 30/04/2020 at a quarter-hourly basis . . . . .	128
4.31	Schematic representation of reducing the consumption at peaks and load-shifting. . . . .	129
4.32	Wind turbines capacity factor during the month of April, 2017 in Froyennes . . . . .	130
4.33	PV panels capacity factor in Tournai, Belgium 2017 . . . . .	131
4.34	Samples of the weekly load consumption of <i>Quartier Negundo</i> . . . . .	132
4.35	illustration of the $\mu$ -PHES prototype in the <i>Quartier Negundo</i> (left) and a second case study (right) . . . . .	133
4.36	Discounted cash flow and NPV of the $\mu$ -PHES in the <i>Quartier Negundo</i> (left) and in the second case study (right). . . . .	137
4.37	Estimations of the NPV of the $\mu$ -PHES of two further scenarios: without VFD but coupled with a gear-box and with a progressive electricity price raise (+ 0.5%/year) . . . . .	137
4.38	LCOE sensitivity analysis using the $\mu$ -PHES in <i>Quartier Negundo</i> . . . . .	138
4.39	LCOE comparison of the $\mu$ -PHES with other storage technologies . . . . .	139
5.1	Numerical investigation workflow and used tools in the methodology . . . . .	143
5.2	Overview of the simulated domain . . . . .	144

5.3	Detail of the geometry in CATIA V5 R25 for the baseline cutwater and a tested configuration . . . . .	145
5.4	Domain geometry improvements by rebuilding the cutwater . . . . .	145
5.5	Flow path in a centrifugal pump volute . . . . .	148
5.6	Qualitative representation of a volute with a tangential exit . . . . .	148
5.7	Variation of the cutwater rounding variable $R$ . . . . .	150
5.8	Variation of the cutwater length variable $S$ . . . . .	150
5.9	Variation of the cutwater tilt angle variable $A$ . . . . .	150
5.10	Velocity triangles at the entrance and exit of the PaT runner . . . . .	151
5.11	Blade to blade topology map of the mesh blocks and cell point distributions . . . . .	151
5.12	Full mesh illustration of the pump impeller and blade to blade topology	152
5.13	View of the wall surfaces of a volute and cut-view of a volute mesh .	153
5.14	Mesh convergence test conducted to assess the independence of the grid on the accuracy of the solution. . . . .	153
5.15	Evaluation of non-dimensional wall distance $y^+$ for the volute and impeller in the baseline case. . . . .	154
5.16	Examples of fitted regression of two-variables relationship . . . . .	158
5.17	Numerical simulations and experimental data comparison for the pump specific energy coefficient $\Psi$ and pump efficiency $\eta_h$ over the relative discharge number. . . . .	161
5.18	Numerical and experimental comparison for the PaT specific energy coefficient $\Psi$ and PaT efficiency $\eta_h$ over the relative discharge number.	161
5.19	Flow field for the baseline cutwater at $\varphi_{BEP}$ . . . . .	162
5.20	Illustrations of three different cutwater at $\varphi = 0.0152$ . . . . .	163
5.21	Head and efficiency characteristics for cutwaters with $S = 2L$ as a function of the discharge number and the cutwater angle, $A$ . . . . .	164
5.22	Head and efficiency characteristics for cutwaters with $S = 3L$ as a function of the discharge number and the cutwater angle, $A$ . . . . .	165
5.23	Head and efficiency characteristics for cutwaters with $S = 4L$ as a function of the discharge number and the cutwater angle, $A$ . . . . .	166
5.24	Numerical error for mass-flow mismatch between the inlet and outlet over the relative discharge number $\varphi/\varphi_{BEP}$ for 224 simulations. . . . .	167
5.25	Test model accuracy for $\hat{\eta}_{PAT} = f(S, R, A, Q_{11}, n_{11})$ . . . . .	169
5.26	Test model accuracy for $\hat{Q}_{11} = g(S, R, A, n_{11})$ . . . . .	170
5.27	$\Psi$ and $\eta$ of the baseline and optimal cutwater designs at $N_{PAT}$ . . . . .	172
5.28	Velocity streamline and velocity magnitude [m/s] contour for the baseline geometry and the optimal asset at different discharge number	173
5.29	Static pressure profiles at mid-flow span and $C_p$ contour for the rotor/stator interface in both baseline and optimized cutwater geometry.	174
5.30	Contours of the absolute velocity angle [rad] across the volute cut-view and on the R/S used in Fig. 5.29. . . . .	175
5.31	Contours of the turbulence kinetic energy $k$ distribution [m <sup>2</sup> /s <sup>2</sup> ] across the volute cut-view. . . . .	175

5.32	3D overview of the absolute velocity vector profiles [m/s] in the baseline geometry. . . . .	176
5.33	3D overview of the absolute velocity vector profiles [m/s] in the optimized geometry. . . . .	176
5.34	Through-flow vorticity contours over the volute cross-sections at BEP for the baseline and optimized geometry. . . . .	177
5.35	RANS and URANS characteristics of $\Lambda_{opt}$ in PaT mode. . . . .	178
5.36	URANS $\alpha_2$ fluctuations in points P1, P2, P3, and P4. The coordinates $(x, y)$ of stations exhibited in Figure are as follows: P1(0, 0.162), P2(0.162, 0), P3(0, -0.162), and P4(-0.162, 0). . . . .	179
5.37	URANS $\alpha_2$ fluctuations in points at different span angles . . . . .	179
5.38	Comparison of $\alpha_2$ for steady and unsteady states at $\varphi/\varphi_{BEP} = 0.87$ . . . . .	180
5.39	Comparison of $\alpha_2$ for steady and unsteady states at $\varphi/\varphi_{BEP} = 0.77$ . . . . .	181
5.40	Comparison of $\alpha_2$ for steady and unsteady states at $\varphi/\varphi_{BEP} = 0.97$ . . . . .	182
5.41	PaT performances with the baseline and optimal cutwater designs for $\Omega = [0.85, 1.28]$ . . . . .	183
5.42	Detail of the PaT efficiency in Fig. 5.41 for the optimized geometry under speed variations $\Omega$ . . . . .	184
5.43	CFD velocity streamline of the pump . . . . .	186
5.44	Absolute pressure distribution along the volute cross-section for five representative cases. . . . .	187
5.45	CFD velocity streamline and $C_p$ contour of pump cutwater . . . . .	188
5.46	Pump hydraulic efficiency at $\varphi/\varphi_{BEP} = 1$ . . . . .	189
5.47	Pump hydraulic efficiency at $\varphi/\varphi_{BEP} = 0.9$ . . . . .	190
5.48	Pump hydraulic efficiency at $\varphi/\varphi_{BEP} = 1.1$ . . . . .	190
5.49	Comparison of the numerical pump characteristics with the baseline geometry and $\Lambda_{opt}$ . . . . .	191
5.50	RANS velocity magnitude iso-surface ( $V = 2$ m/s) downstream the cutwater of $\Lambda_{opt}$ in pump mode. . . . .	192
5.51	Water flow streamlines of the pump at $\varphi/\varphi_{BEP} = 1$ under $\Lambda_{opt}$ asset. . . . .	192
5.52	URANS velocity magnitude iso-surface ( $V = 1$ m/s) downstream the cutwater of $\Lambda_{opt}$ in pump mode. . . . .	193
5.53	URANS velocity magnitude iso-surface ( $V = 2$ m/s) downstream the cutwater of $\Lambda_{opt}$ in pump mode. . . . .	193
6.1	Achievements of the PhD thesis. . . . .	198
A.1	Slip factor effect on velocity triangle . . . . .	215
A.2	Schematic representations of the overlap angle. . . . .	217
A.3	Trailing edge parameters of the pump . . . . .	218
A.4	Single-arc method of constructing blade profile . . . . .	219
B.1	Static head of the pump minimum pressure point . . . . .	222
B.2	Critical $\sigma_c r$ for pumps, Kaplan turbines, Francis turbines and PaTs . . . . .	223



# List of Tables

2.1	Available turbomachinery solutions for PHES and UPSH . . . . .	21
2.2	Pumped storage plants using Deriaz pump-turbine. . . . .	29
2.3	Performance prediction methods for PaT. . . . .	35
3.1	Energy infrastructure trends summary . . . . .	42
3.2	Energy tariff in the Amazon region (US\$/kWh) in 2018 [National Electric Energy Agency 2018]. . . . .	50
3.3	Existing pumped hydro storage plants in Belgium. . . . .	54
3.4	Breakdown of the total initial investment cost expressed in k€ . . . .	58
3.5	Data and NPV results for a preliminary economic evaluation of UPSH in Martelange. . . . .	61
3.6	Mechanical data of Deriaz prototype working in Naussac II. . . . .	75
3.7	Operational data in pumping and generating Deriaz prototype in Naussac II power-plant . . . . .	75
3.8	Mechanical data of the downsized Deriaz pump. In Figure a down- sized model of a Deriaz pump-turbine . . . . .	76
3.9	Mesh convergence test on the same project case at $\delta\beta = 5^\circ$ , $a_0 = 56\%$ and mass flow of $0.18 \text{ [m}^3/\text{s]}$ . . . . .	78
3.10	Mesh quality summary for turbine case at $\delta\beta = 5^\circ$ and closing runner blade at $\delta\beta = 0^\circ$ . . . . .	79
3.11	Turbulent model test for an assigned mesh . . . . .	80
3.12	Adopted boundary conditions for the CFD simulations. . . . .	82
3.13	Inlet condition for the flow direction in turbine mode. . . . .	83
4.1	Extract of the pump data-sheet Main parameters of the pump in- stalled at the micro pumped storage facility plant. . . . .	106
4.2	Instrumentation range and accuracy summary. . . . .	108
4.3	Summary of PHES control and command actions. . . . .	113
4.4	Value for green certificate in Brussels, Belgium, from June 2020. . . .	125
4.5	List of the main players in the electricity market in Belgium. . . . .	126
4.6	Averaged capacity factor for the PV panels and wind turbines in Belgium 2017. . . . .	130
4.7	Capital costs for $\mu$ -PHES in <i>Quartier Negundo</i> and cost estimation for the second case study. . . . .	134
4.8	Data and NPV results for a economic evaluation of $\mu$ -PHES. . . . .	137
5.1	List of independent variables describing the cutwater. . . . .	149
5.2	Mesh details for the baseline case study domain . . . . .	154
5.3	Turbulent model test applied to a PaT: Spalart-Allmaras (SA), $k-\varepsilon$ models, $k-\omega$ with or without Extended Wall Function (EWF). . . . .	156

5.4	Adopted boundary conditions for the CFD simulations. . . . .	157
5.5	Basis functions for $\hat{\eta}_{PAT}$ by the variables S, R, A $Q_{11}$ , and $n_{11}$ . . . . .	169
5.6	Basis functions for $\hat{Q}_{11}$ by the variables S, R, A, and $n_{11}$ for third degree maximum fit spline. . . . .	170
5.7	Predicted optimal cutwater geometry and the resulting efficiency by varying $n_{11}$ . . . . .	171
5.8	Divergence of the absolute velocity angle $\alpha_2$ in RANS and URANS. . . . .	179
6.1	SWOT analysis of $\mu$ -PHES. . . . .	204
A.1	Mechanical data of Deriaz prototype working in Naussac II. . . . .	212
A.2	Design values along five main streamlines. . . . .	216

# Nomenclature

## Latin symbols

$a_0$	Guide vane opening	mm
$A$	Cutwater inclination angle	deg
$b$	Passage depth	m
$B$	Systematic error	-
$c$	Absolute velocity	m/s
$c_{mix}$	Celerity of the sound in a fluid mixture	m/s
$C$	Cost	€
$C_d$	Discharge coefficient	-
$C_p$	Static pressure coefficient	-
$d$	Yearly discount rate	-
$dx$	Average element size	m
$D$	Diameter	m
$e$	Specific hydraulic energy	J/kg
$E$	Energy	kWh
$f$	Grid frequency	Hz
$g$	Gravitational acceleration	m/s <sup>2</sup>
$h$	Head ratio	-
$h_i$	Hinge function factor	-
$H$	Head	m
$i$	Incidence deviation	deg
$k_i$	Constant	-
$K$	Project duration	year
$L$	Length	m
$m$	Mass flow rate	kg/s
$M$	Torque	Nm
$n$	Rotational frequency	rot/s
$np$	Number of poles in a generator	-
$n_{11}$	Unit speed	-
$N$	Rotational speed	rpm
$N_s$	Specific speed	rpm, m <sup>3</sup> /s, m
$p$	Pressure	Pa
$P$	Power	kW
$q$	Discharge ratio	-
$Q$	Discharge	m <sup>3</sup> /s
$Q_{11}$	Unit discharge	-
$r$	Nominal cutwater radius	mm
$R$	Radius	m
$Re$	Reynolds number	-

---

$S$	Cutwater stretching	m
$t$	Time	s
$u$	Peripheral velocity	m/s
$U$	Uncertainty	-
$v$	General speed vector	m/s
$V$	Volume	m <sup>3</sup>
$V_r$	Energy loss fraction	-
$w$	Relative velocity	m/s
$W$	Direction flow	-
$y^+$	Non-dimensional wall distance	-
$z$	Number of blades	-
$Z$	Altitude	m

### Greek symbols

$\alpha$	Absolute velocity angle	deg
$\beta$	Relative velocity angle	deg
$\gamma$	Blade pivot angle	deg
$\eta$	Efficiency	-
$\Theta$	Angular coordinate	deg
$\lambda$	Blade inclination	deg
$\Lambda$	Domain	-
$\mu$	Dynamic viscosity	Pa s
$\nu$	Dimensionless turbine specific speed	-
$\xi$	Efficiency ratio	-
$\pi$	Power number	-
$\rho$	Density	kg/m <sup>3</sup>
$\sigma$	Thoma number	-
$\tau$	Slip coefficient	-
$\varphi$	Discharge number	-
$\psi$	Correction coefficient	-
$\omega$	Angular speed	rad/s
$\Omega$	Rotational speed ratio	-
$\Psi$	Specific energy coefficient	-



---

**Subscripts**

<i>11</i>	Unit factor
<i>1</i>	Pump inlet
<i>2</i>	Pump outlet
<i>a</i>	Available
<i>ad</i>	Normalised value
<i>CFD</i>	Numerical
<i>cr</i>	Critical
<i>curt</i>	Curtailement
<i>exp</i>	Experimental
<i>eff</i>	Effective
<i>fit</i>	Fitting
<i>g</i>	Geodetic
<i>geo</i>	Geometrical
<i>h</i>	Hydraulic
<i>i</i>	Index
<i>in</i>	Injected
<i>l</i>	Loss
<i>L</i>	Liquid
<i>m</i>	Mechanic
<i>m</i>	Meridional
<i>max</i>	Maximum
<i>min</i>	Minimum
<i>md</i>	Model
<i>n</i>	Nominal
<i>opt</i>	Optimal
<i>p</i>	Pump
<i>pty</i>	Prototype
<i>r</i>	Radial
<i>rw</i>	Runaway
<i>syt</i>	System
<i>t</i>	Turbine
<i>th</i>	Theoretic
<i>u</i>	Circumferential direction
<i>v</i>	Volumetric
<i>vp</i>	Vapour
<i>vol</i>	Volumetric

---

### Acronyms

AGV	Adjustable Guide Vanes
AR	Aspect Ratio
ATM	Aero-Thermo-Mechanics
BEP	Best Efficiency Point
BF	Basis Function
CAD	Computer Aided Design
CFD	Computational Fluid Dynamic
CFL	Courant-Friedrich-Levy number
CPU	Central Processing Unit
DES	Decentralized energy sources
DNIT	National Department of Transport Infrastructure
DSO	Distribution system operator
EEX	European Energy Exchange
EES	Electrical Energy Storage
ER	Expansion Ratio
EWf	Extended Wall Function
FEA	Finite Element Analysis
FS	Factor of safety
HGS	Hydraulic Gravity Storage
IDETA	Agence de Développement Territorial
LCOE	Levelised Cost Of Energy
LCOS	Levelized Cost of Storage
MARS	Multivariate Adaptive Regression Spline
MTBF	Mean Time Between Failures
NPV	Net Present Value
OPEX	Operating Expense
O&M	Operations and Maintenance
PaT	Pump as Turbine
PHES	Pumped Hydro Energy Storage
$\mu$ -PHES	Micro Pump Hydro Energy Storage
PLC	Programmable Logic Controller
PV	PhotoVoltaic
RES	Renewable Energy Source
RPT	Reversible Pump-Turbine
SA	Spalart-Allmaras model
SPS	Seawater Pump Storage
SST	Shear Stress Transport
TSO	Transmission System Operator
UPHS	Underground Pumped-Storage Hydroelectricity
UPS	Uninterruptible Power Supply
VFD	Variable Frequency Drive